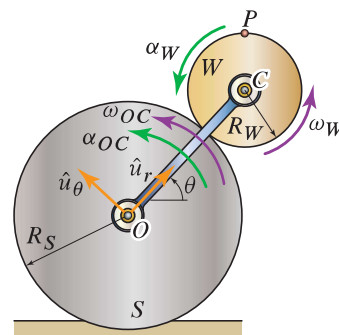


Problem 6.121

A wheel W of radius $R_W = 2$ in. rolls without slip over the stationary cylinder S of radius $R_S = 5$ in., and the wheel is connected to point O via the arm OC .

Determine the acceleration of the point on the wheel W that is in contact with S for $\omega_{OC} = 7.5$ rad/s and $\alpha_{OC} = 2$ rad/s².


Solution

Relating the velocity of C to that of O , we obtain

$$\begin{aligned}\vec{v}_C &= \vec{v}_O + \vec{\omega}_{OC} \times \vec{r}_{C/O} = \omega_{OC} \hat{k} \times (R_S + R_W) \hat{u}_r \\ &= (R_S + R_W) \omega_{OC} \hat{u}_\theta,\end{aligned}$$

where we have used the fact that O is a fixed point. Now let Q be the point on W in contact with S . Since W rolls without slip on S , $\vec{v}_Q = \vec{0}$, and so

$$\vec{v}_C = \vec{v}_Q + \vec{\omega}_W \times \vec{r}_{C/Q} = \omega_W \hat{k} \times R_W \hat{u}_r = R_W \omega_W \hat{u}_\theta.$$

Equating these two expressions for \vec{v}_C , we obtain the angular velocity of the wheel as

$$(R_S + R_W) \omega_{OC} \hat{u}_\theta = R_W \omega_W \hat{u}_\theta \quad \Rightarrow \quad \omega_W = \frac{R_S + R_W}{R_W} \omega_{OC}. \quad (1)$$

Relating the acceleration of C to that of O , we have

$$\begin{aligned}\vec{a}_C &= \vec{a}_O + \vec{\alpha}_{OC} \times \vec{r}_{C/O} - \omega_{OC}^2 \vec{r}_{C/O} = \alpha_{OC} \hat{k} \times (R_S + R_W) \hat{u}_r - \omega_{OC}^2 (R_S + R_W) \hat{u}_r \\ &= -(R_S + R_W) \omega_{OC}^2 \hat{u}_r + (R_S + R_W) \alpha_{OC} \hat{u}_\theta, \quad (2)\end{aligned}$$

where we have used the fact that point O is fixed. Now relating the acceleration of Q to that of C , we obtain

$$\begin{aligned}\vec{a}_Q &= \vec{a}_C + \vec{\alpha}_W \times \vec{r}_{Q/C} - \omega_W^2 \vec{r}_{Q/C} \\ &= -(R_S + R_W) \omega_{OC}^2 \hat{u}_r + (R_S + R_W) \alpha_{OC} \hat{u}_\theta + \alpha_W \hat{k} \times (-R_W \hat{u}_r) - \omega_W^2 (-R_W \hat{u}_r) \\ &= [-(R_S + R_W) \omega_{OC}^2 + R_W \omega_W^2] \hat{u}_r + [(R_S + R_W) \alpha_{OC} - R_W \alpha_W] \hat{u}_\theta.\end{aligned}$$

where we have used Eq. (2) for \vec{a}_C . Since W is rolling without slip over S , the component of acceleration tangent to the two bodies at Q must be equal. Therefore, since S is stationary, it must be true that $a_{Q\theta} = 0$, and therefore, the acceleration of A is given by

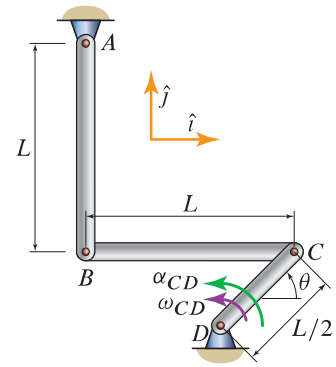
$$\vec{a}_Q = \left[-(R_S + R_W) \omega_{OC}^2 + \frac{(R_S + R_W)^2}{R_W} \omega_{OC}^2 \right] \hat{u}_r = \frac{R_S}{R_W} (R_S + R_W) \omega_{OC}^2 \hat{u}_r,$$

where Eq. (1) was used for ω_W . Using $R_W = 2$ in. = $\frac{2}{12}$ ft, $R_S = 5$ in. = $\frac{5}{12}$ ft, and $\omega_{OC} = 7.5$ rad/s, we have

$$\vec{a}_Q = 82.03 \hat{u}_r \text{ ft/s}^2.$$

Problem 6.125

At the instant shown, bar CD is rotating with an angular velocity 20 rad/s and with angular acceleration 2 rad/s^2 in the directions shown. Furthermore, at this instant $\theta = 45^\circ$. If $L = 2.25 \text{ ft}$, determine the angular accelerations of bars AB and BC .



Solution

First finding the velocity of C , we obtain

$$\vec{v}_C = \vec{\omega}_{CD} \times \vec{r}_{C/D} = \omega_{CD} \hat{k} \times \frac{1}{2}L(\cos \theta \hat{i} + \sin \theta \hat{j}) = \frac{1}{2}L\omega_{CD}(-\sin \theta \hat{i} + \cos \theta \hat{j}),$$

where we have enforced the fact that point D is fixed. Now relating the velocity of B to points A and C , we obtain

$$\begin{aligned} \vec{v}_B &= \vec{\omega}_{AB} \times \vec{r}_{B/A} = \vec{v}_C + \vec{\omega}_{BC} \times \vec{r}_{B/C} \\ \Rightarrow \omega_{AB} \hat{k} \times (-L \hat{j}) &= \frac{1}{2}L\omega_{CD}(-\sin \theta \hat{i} + \cos \theta \hat{j}) + \omega_{BC} \hat{k} \times (-L \hat{i}) \\ \Rightarrow L\omega_{AB} \hat{i} &= -\frac{1}{2}L\omega_{CD} \sin \theta \hat{i} + \left(\frac{1}{2}L\omega_{CD} \cos \theta - L\omega_{BC}\right) \hat{j}. \end{aligned}$$

This equation represents the following two scalar equations in the unknowns ω_{AB} and ω_{BC} :

$$L\omega_{AB} = -\frac{1}{2}L\omega_{CD} \sin \theta \quad \text{and} \quad 0 = \frac{1}{2}L\omega_{CD} \cos \theta - L\omega_{BC},$$

the solution of which is

$$\omega_{AB} = -\frac{1}{2}\omega_{CD} \sin \theta \quad \text{and} \quad \omega_{BC} = \frac{1}{2}\omega_{CD} \cos \theta.$$

Proceeding similarly with the acceleration analysis, we begin by finding the acceleration of point C as

$$\begin{aligned} \vec{a}_C &= \vec{\alpha}_{CD} \times \vec{r}_{C/D} - \omega_{CD}^2 \vec{r}_{C/D} = \alpha_{CD} \hat{k} \times \frac{1}{2}L(\cos \theta \hat{i} + \sin \theta \hat{j}) - \omega_{CD}^2 \left[\frac{1}{2}L(\cos \theta \hat{i} + \sin \theta \hat{j})\right] \\ \Rightarrow \vec{a}_C &= -\frac{1}{2}L(\alpha_{CD} \sin \theta + \omega_{CD}^2 \cos \theta) \hat{i} + \frac{1}{2}L(\alpha_{CD} \cos \theta - \omega_{CD}^2 \sin \theta) \hat{j}. \end{aligned}$$

We then relate the acceleration of B to points A and C using

$$\vec{a}_B = \vec{\alpha}_{AB} \times \vec{r}_{B/A} - \omega_{AB}^2 \vec{r}_{B/A} = \vec{a}_C + \vec{\alpha}_{BC} \times \vec{r}_{B/C} - \omega_{BC}^2 \vec{r}_{B/C},$$

or, substituting in known quantities, we obtain

$$\begin{aligned} \alpha_{AB} \hat{k} \times (-L \hat{j}) - \frac{1}{4}\omega_{CD}^2 \sin^2 \theta (-L \hat{j}) \\ = -\frac{1}{2}L(\alpha_{CD} \sin \theta + \omega_{CD}^2 \cos \theta) \hat{i} + \frac{1}{2}L(\alpha_{CD} \cos \theta - \omega_{CD}^2 \sin \theta) \hat{j} \\ + \alpha_{BC} \hat{k} \times (-L \hat{i}) - \frac{1}{4}\omega_{CD}^2 \cos^2 \theta (-L \hat{i}), \end{aligned}$$

where we have used the expressions for ω_{AB} and ω_{BC} determined above. Equating components, we have two equations for α_{AB} and α_{BC} :

$$L\alpha_{AB} = -\frac{1}{2}L(\alpha_{CD}\sin\theta + \omega_{CD}^2\cos\theta) + \frac{1}{4}L\omega_{CD}^2\cos^2\theta,$$

$$\frac{1}{4}L\omega_{CD}^2\sin^2\theta = \frac{1}{2}L(\alpha_{CD}\cos\theta - \omega_{CD}^2\sin\theta) - L\alpha_{BC},$$

the solution of which is

$$\alpha_{AB} = \frac{1}{4}\omega_{CD}^2\cos^2\theta - \frac{1}{2}\alpha_{CD}\sin\theta - \frac{1}{2}\omega_{CD}^2\cos\theta,$$

$$\alpha_{BC} = -\frac{1}{4}\omega_{CD}^2\sin^2\theta + \frac{1}{2}\alpha_{CD}\cos\theta - \frac{1}{2}\omega_{CD}^2\sin\theta.$$

Using $\omega_{CD} = 20$ rad/s, $\alpha_{CD} = 2$ rad/s², $\theta = 45^\circ$, and $L = 2.25$ ft, and expressing our answer in vector form, we have

$$\vec{\alpha}_{AB} = -92.13 \hat{k} \text{ rad/s}^2 \quad \text{and} \quad \vec{\alpha}_{BC} = -190.7 \hat{k} \text{ rad/s}^2.$$